

工學碩士 學位論文

A Study on the Development
of the Software of Ship Hull Stress Monitoring System

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Abstract

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A Study on the Development of Ship Hull Stress Monitoring System

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Abstract

In the recent years, major ship registers have demanded improved safety on the hull stress of large bulk carriers which are on navigation or cargo handling in harbour.

Under these circumstances, a system that monitors hull stress and ship condition is becoming more and more important. If efficient and appropriate navigational information is given, safety of navigation would be greatly improved.

The major ship registers of the globe are investing a great effort on the development of a system that monitors the hull stress of ship. Using this system, information of hull stress and ship motion is given to the users and also the data is stored on the external data storage system simultaneously.

Through this study, a software that monitors hull stress was developed. Not only can randomized input-data of the standard hardwares be applied to the system, but also this system can be operated on and applied to real hardware systems.

가 . 가

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,

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.

,

가 ,

,

(International Maritime Organization : IMO)

20,000DWT (Deadweight Ton : DWT) Bulk Carrier

(Hull Stress Monitoring System : HSMS)

[1], (Lloyd's Register : LR),

(Det norske Veritas : DnV), (American Bureau of Shipping : ABS) HSMS

HSMS [2].

, , HSMS

. , HSMS 가 가 .

(Korean Register of Shipping : KR),

가 .

가 HSMS

, ,

Record : VDR)

(Voyage Data

.

HSMS

2.1

HSMS

IMO HSMS

LR (Hull Surveillance System) , DnV

LR (Hull Condition Monitoring System) , ABS [3],

(KR)

[4]. , IMO DWT 20,000 (Hull Stress Monitoring System)

Table 1, Table 2 Table 3 .

Table 1

Table 1 , $\pm 5 \mu\epsilon$, 0 5Hz ,

가 0.01L ,

$\pm 2g$ $\pm 0.1g$,

. Table 2 ,

. Table 3 ,

, HSMS

. Table 3 ,

. 가 (+ + +) ,
 가 (+ + + +) VDR , LVDT (Linear Variable
 Differential Transformer)
 Table 1 Table 2 ,

Table 1 Sensor

	Requirements
LR	① (2) - - : $\pm 5 \mu\epsilon$ - : 0 5Hz ② 가 (1) - : , 0.01L . - : $\pm 2g$ - : $\pm 0.1g$
DnV	① (2) - - : $\pm 5 \mu\epsilon$ - : 0 5Hz ② 가 (1) - : , 0.01L . - : $\pm 20m/ sec^2$ - : $\pm 0.1g$
ABS	① - - : $\pm 5 \mu\epsilon$ - : 0 5Hz ② 가 - : , 0.01L . - : $\pm 2g+1g$ - : $\pm 0.01g(0.5\%)$
KR	① - - : $\pm 5 \mu\epsilon$ - : 0 5Hz ② 가 - : , 0.01L . - : $\pm 1g+1g$ - : $\pm 0.01g(0.5\%)$
IMO	① () - , ② 가 ③ Roll Sway 가 ()

Table 2 Data Processing and Display

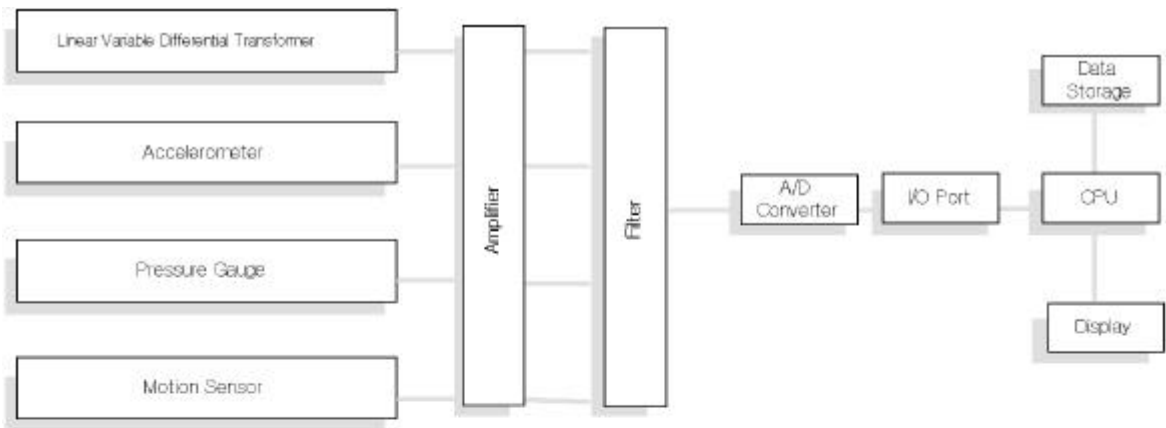
	Requirements
LR	① , , 5 30 ② slam (counting) ③ 4 trend ④ bridge ⑤ 30 / , , , zero cross , slam , ,
DnV	① 5 30 ② 24 trend ③ ④ bridge , , , histogram counting ⑤ 30 / , , , zero corss , slam , ,
ABS	① 20 ② 가 ③ ④ bridge , , , histogram counting ⑤ 20 / , , , zero corss , slam , ,
KR	① 1 5 ② 1 trend ③ ④ bridge , , / , zero cross , ⑤ 5 , , / , zero cross ,
IMO	① , () ② ③ . ④ , 가

Table 3 Equipment Position and Type

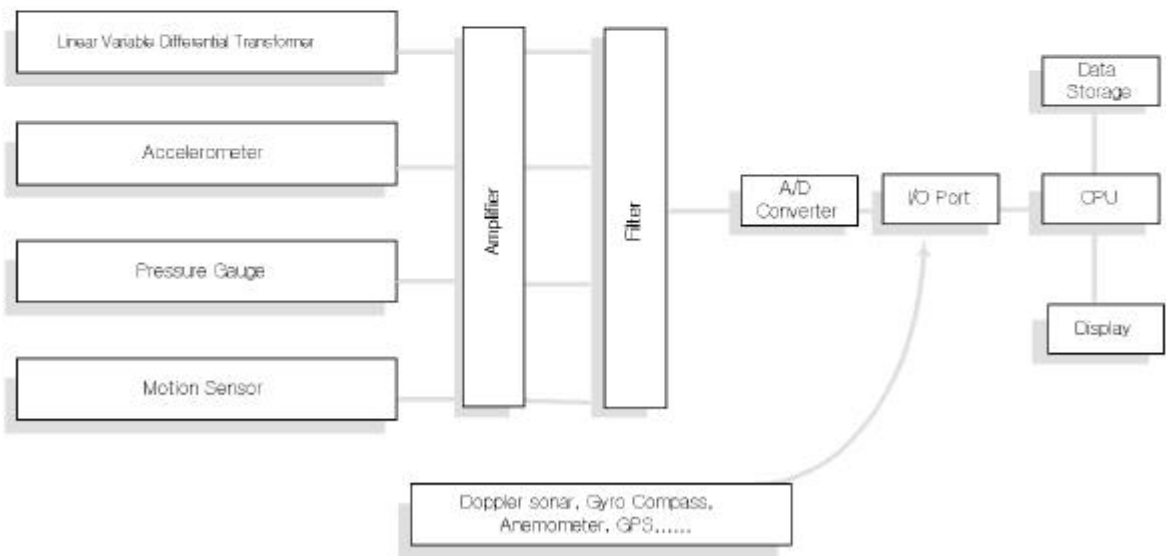
measuring object	equipment position	equipment type
<p align="center">① Hull Girder Stress</p>	LVDT : 4 - 2 LVDT : Midship P&S side - 1 LVDT : L/4 from Bow - 1 LVDT : L/4 form stern	Hull girder stress
<p align="center">② Ship Motion</p>	Clinometer : 2 - 2 Clinometer : Deckhouse	Pitch and roll angle
<p align="center">③ Slamming</p>	Accelerometer and/or Pressure gauge - 1 Accelerometer : Bow - 1 Pressure gauge : Bow btm	Acceleration
<p align="center">Data File Storage</p>	PC External Cartridge Drive - system rack mounted on deck house	trend of hull girder stress and impact response by bow slamming
<p align="center">Crack</p>	Strain gauge : 2 and - 2 strain gauge : midship on bilge at port/stbd side - interfacing PC with GPS, Speed Log, Torque meter and Loading computer	Local stress for evaluation of initial fatigue. Voyage condition

2.2 HSMS

Fig. 1 . Fig. 1(a)
 , slamming 가
 ,
 가 A/D Converter
 /
 Fig. 1(b) Fig. 1(a) GPS(Global
 Positioning System), Doppler sonar log, Gyro compass Anemometer
 가 VDR 가 .
 VDR
 ,
 가 .



(a)



(b)

Fig. 1 Diagram of HSMS

3

3.1

LVDT

가 가

Table 4

[5].

Table 4 Scale and Precision of Equipment

position & objects	equipment	full scale	frequency range	position	precision
Ship Motion (Heave, Sway, Surge)	Accelerometer	$\pm 2g + 1g$	5Hz	Midship	$\pm 0.01g$
Ship Motion (Pitch, Roll)	Angle Sencor (Clinometer)	$\pm 30^\circ$	5Hz	Midship or Appropriate Position	$\pm 0.3^\circ$
Bow Acceleration	Accelerometer	$\pm 2g + 1g$	5Hz	Forward end	$\pm 0.01g$
Hull Girder Stress	LVDT	$\pm 2.5mm$	5Hz	Midship on Deck	$\pm 2\%$ ($\pm 5 \mu$)
Local Stress	Strain Gauge	$\pm 2500 \mu$	5Hz	Midship in way of Longitudinal	$\pm 5 \mu$
Pressure	Pressure Gauge	900 1100 mbar	5Hz	Bow on Bottom	$\pm 5\%$

3.1.1 LBSG(Long Based Strain Gauge)

LBSG

rod, 2, 3
(Fig. 2) [6].

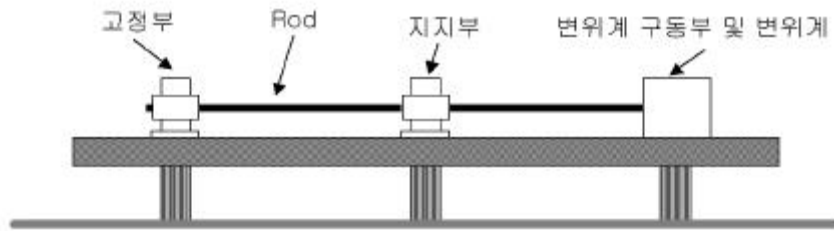


Fig. 2 Diagram of LBSG

LBSG

LVDT Linear Potentiometer(LPM)
가

. LVDT

. LPM

가

LPM

(Intrinsically Safe)

, LVDT

가

[7].

MIDORI

LP- 10FB

Table 5

Table 5 Electrical Specification of LP- 10FB

	10 mm
	0.1, 0.2, 0.5, 1 k Ω
	± 20 %
	± 1 %
	0.3 W / 70
	0.1 %
	100 M Ω DC 500 V
	AC 500 V 1
	± 400 ppm /

3.1.2 가

가 1 가
 3 가 가
 가 가
 servo-type, ,
 bridge-type
 piezo-type . ± 2g+1g 가
 가 .
 가 .
 . 0 0
 . 900
 1100mbar .

3.2

Fig. 3 .
 multiplexer PC Multiplexer
 PC RS-232C .

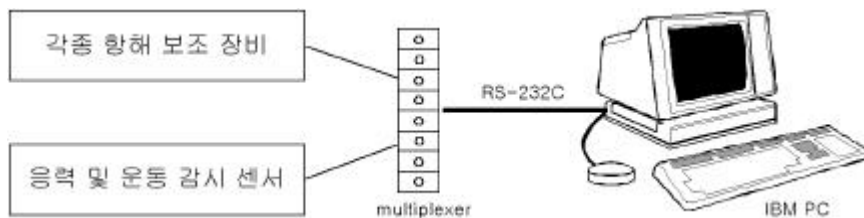


Fig. 3 Diagram of the Interface

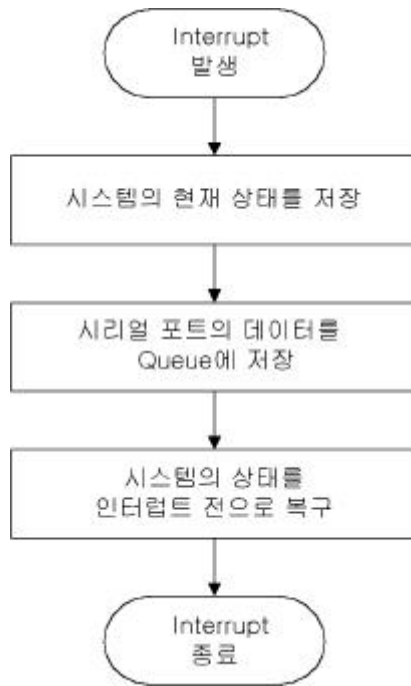


Fig. 4 Flowchart of the Interrupt Function

Fig. 4 Function . Queue , 가 Queue
 가 Queue
 Queue
 . Queue 가 buffer
 가 가 data buffer Fig. 5
 . data buffer

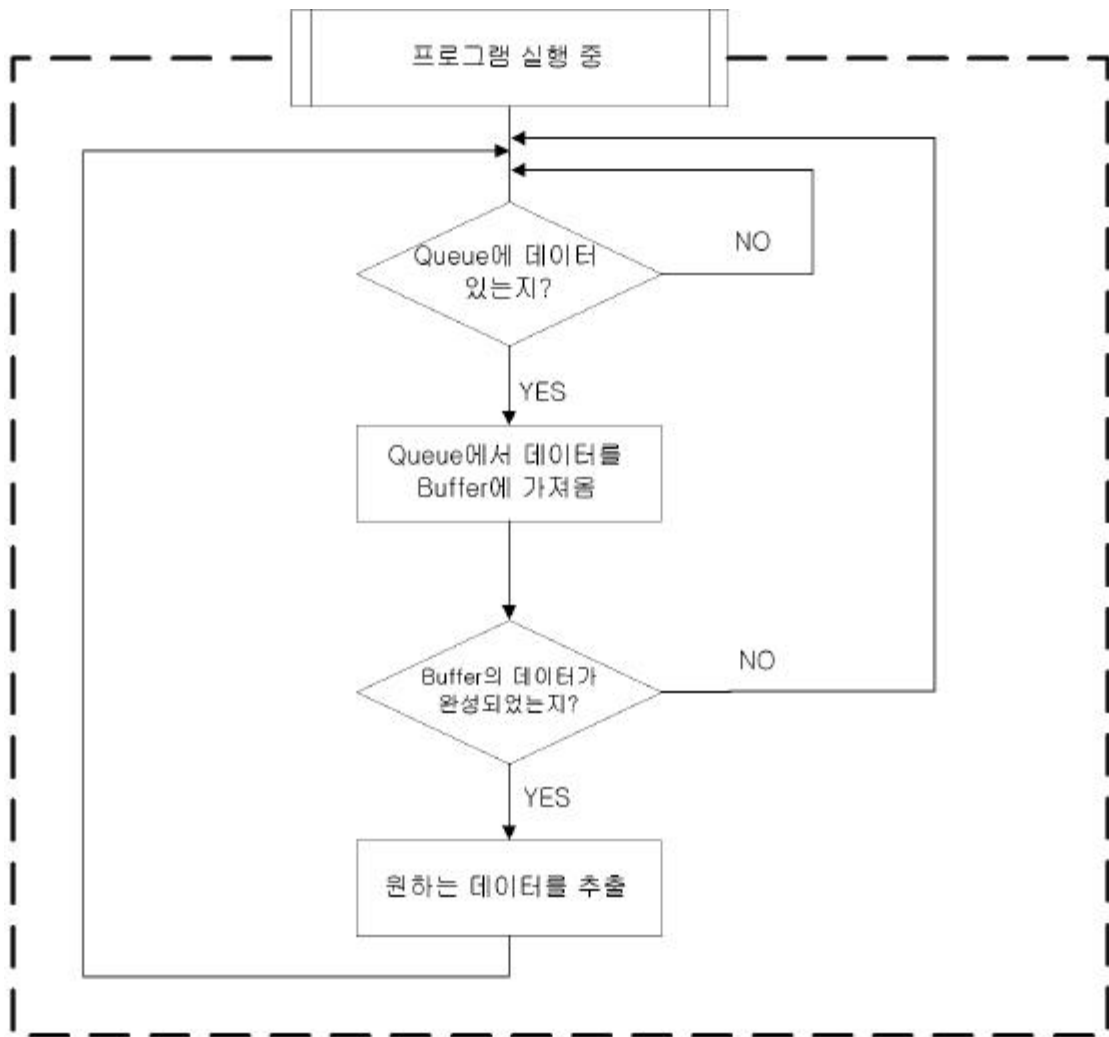


Fig. 5 Flowchart of Data Processing

Fig. 5

User Interface : GUI) Fig. 5 GUI(Graphical
 , HSMS
 , 가
 (Multi-tasking) , 가
 GUI .

4

HSMS

가 , Table 2 Real Time Multi-tasking , [8].

4.1

5 , rms(root mean square) zero-cross , 가 .

Table 6 .

(1)

$$M_{total} = \frac{M_{still} + M_{wave}}{M_t} \times 100 (\%) \quad :$$

$$M_{total} = \frac{M_{still}}{M_s} \times 100 (\%) \quad :$$

(1)

M_{wave} , M_{total} , M_{still} , M_t Table 6
 M_s , M_w
 LBSG x M_x , M_{still}
 M_{wave} M_x (2) [5].

$$M_x = \left(\frac{V_{out}}{V_{max}} \right) \left(\frac{\Delta \ell_{max}}{\ell} \right) \cdot E \cdot Z_x \quad (2)$$

V_{out} LBSG, V_{max} LBSG
 $\Delta \ell_{max}$ LBSG Rod, ℓ LBSG Rod, E
 (Young's Modulus), Z_x
 M_{still} 5 M_x , (3)

$$M_{still} = \frac{1}{N} \sum M_x \quad (3)$$

N 5
 M_{wave} M_x 2.65, (4)

$$M_{wave} = 2.65 \times \sqrt{\frac{1}{N} \sum (M_x - \bar{M}_x)^2} \quad (4)$$

2.65 30
 [9].
 M_{total} M_{still} M_{wave} , (1) M_{total}

(M_t)

가

,

Table 6 Requirements concerning Strength of Ship

	Requirements
LR	$M_t = M_s + M_w$
	< >
	$M_w = f_1 f_2 M_{w0} \text{ (kN-m)}$
	$f_1 : \text{ship service factor}$
	= -1.1, = 0.5
	$f_2 : \text{wave bending moment factor}$
	= - 1.1 for <i>sagging</i>
	= $\frac{1.9 C_b}{(C_b + 0.7)}$ for <i>hogging</i>
	$C_b \geq 0.6$
	$M_{w0} = 0.1 C_1 C_2 L^2 B (C_b + 0.7)$
	$C_1 = 0.0412L + 4.0 \quad L < 90 \text{ m}$
	= 10.75 - $\left(\frac{300 - L}{100}\right)^{1.5}$ $90 \leq L \leq 300 \text{ m}$
	= 10.75 $300 < L \leq 350 \text{ m}$
	= 10.75 - $\left(\frac{L - 350}{150}\right)^{1.5}$ $350 < L \leq 500 \text{ m}$
	$C_2 : \text{longitudinal distribution factor}$
= 0 for <i>A.P.</i> and <i>F.P.</i>	
= 1.0 for $0.4L \leq x \leq 0.65L$	
< >	
$ M_s = F_d \sigma Z_d \times 10^3 - M_w \text{ (kN-m)}$	
$\sigma : \quad \quad \quad (\quad + \quad) \quad \text{(N/mm}^2\text{)}$	
= $\frac{175}{k_L}$	
$k_L :$	
= $\frac{245}{\sigma_0}$	
0.72, 0.72, 0.72,	
= 1.0	
가	

	Requirements
LR	$\sigma_0 : \quad , 353 \text{ N/mm}^2$ $Z_{\min} : \quad = f_1 k_L C_1 L^2 B (C_b + 0.7) \times 10^{-6} \text{ m}^3$ $F_d : \text{ Local scantling reduction factors}$ $= \frac{\sigma_d}{\sigma}$ $\sigma_d = \frac{\overline{M}_s + M_w}{Z_d} \times 10^{-3}$ $, F_d \quad \sigma_d \quad 0.67, \quad 0.75$ $Z_d :$ <p style="text-align: center;">[LR Rules and Regulations for the Classification of Ships, Part 3 ship Structures, Chapter 4 Longitudinal Strength, 1992]</p>
DnV	$M_t = M_s + 1.1 M_w \quad \text{in intact condition}$ $M_s = C_w L^2 B (0.1225 - 0.015 C_b) \quad \text{kN-m} \quad : \text{ hogging}$ $= - 0.065 C_w L^2 B (C_b + 0.7) \quad \text{kN-m} \quad : \text{ sagging}$ $M_w = 0.19 C_w L^2 B C_b \quad \text{kN-m} \quad : \text{ hogging}$ $= - 0.11 C_w L^2 B (C_b + 0.7) \quad \text{kN-m} \quad : \text{ hogging}$ $C_w = 0.0792 \quad \text{for } L \leq 100 \text{ m}$ $= 10.75 - \{(300 - L)/100\}^{1.5} \quad \text{for } 100 < L < 300 \text{ m}$ $= 0.075 \quad \text{for } 300 \leq L \leq 350 \text{ m}$ $= 10.75 - \{(L - 350)/150\}^{1.5} \quad \text{for } L > 350 \text{ m}$ <p style="text-align: center;">[DnV Hull Structural Design, ships with length 100 meters and above, 1994]</p>

	Requirements
ABS	$M_t = M_s + M_w$ $M_s = C_{st} L^{2.5} B (C_b + 0.5) \quad (\text{ft-1.ton})$ $C_{st} = 0.275 \times 10^{-3} \quad 690 < L \leq 820 \text{ft}$ $C_{st} = \left[0.275 - \frac{L - 820}{1160} \right] \times 10^{-3} \quad 690 < L \leq 820 \text{ft}$ <p style="text-align: center;">$L =$ (ft), $B =$ (ft), $C_b =$ block coefficient</p> $M_w = C_2 L^2 B H_e K_b \quad (\text{ft-1.ton})$ $C_2 = (6.53 C_b + 0.57) \times 10^{-4}$ $H_e = (335 + 4.5 L - 0.00216 L^2) \times 10^{-2} \quad 720 < L \leq 1000 \text{ft}$ $K_b = 1.0 \quad C_b \geq 0.80$ <p style="text-align: center;">[Rules for Building and Classing Steel Vessels, Section 6-Longitudinal Strength, 1977]</p>
KR	<p style="text-align: center;">< ></p> $M_s = \frac{175}{K} Z_{\min} - \alpha M_w$ <p>K :</p> <p>Z_{\min} :</p> $Z_{\min} = C_1 L^2 B (C_b + 0.7) K$ $C_1 = 10.75 - \left(\frac{300 - L}{100} \right) \quad \text{for } 90 \leq L \leq 300 \text{ m}$ $= 10.75 \quad \text{for } 300 \leq L \leq 350 \text{ m}$ $= 10.75 - \left(\frac{300 - L}{100} \right) \quad \text{for } 350 \leq L \leq 500 \text{ m}$ <p>L :</p> <p>B :</p> <p>C_b : (block coefficient)</p> <p>α : , (=0.5), (=1.0)</p>

	Requirements
KR	<p> $M_w :$ $M_w(\text{hogging}) = 0.19 C_1 C_2 L^2 B C_b$ $M_w(\text{sagging}) = - 0.11 C_1 C_2 L^2 B (C_b + 0.7)$ </p> <p> $C_2 :$ $= 1.0$ for $0.4L \leq x \leq 0.65L$ $= 0.0$ for A.P and F.P </p> <p style="text-align: center;">< ></p> <p> $M_w = \frac{175}{K} Z_{\min} - (M_s)_{\text{actual}}$ </p> <p> $(M_s)_{\text{actual}}$ Beam </p> <p style="text-align: right;">가 .</p> <p style="text-align: center;"> $M_{\max} = \frac{W \cdot L}{C}$ </p> <p> $W :$ (ton) $C :$ </p> <p style="text-align: right;"> : 35(hogging), : 32(hogging) : 35(hogging/sagging), : 40(sagging) </p> <p style="text-align: right;">[3 , 1995]</p>

가

가 , 가
가 g 40% [10]. , 가 가 0.4g

(Warning_e)
(σ_{RM_e}) 5.22 (5)

[11].

$$\sigma_{RM_e} = H \sqrt{-\frac{1}{2 \ln \left(\frac{2\pi N_e}{3600R} \right)}} \quad (5)$$

$$Warning_e = \sigma_{RM_e} \times 5.22$$

, H 5
(m), N_e (0m) 가
0.1 . 5.22 10sec

5
R

, (V)
, (6)

$$R = 0.012 \times V + 0.65 \quad (6)$$

, V .

$$Warning_e = 5 \quad , \quad 5$$

$$(5) \quad , \quad 5$$

$$(\sigma_{RMs}) \quad 5.22 \quad (Warning_s) \quad (7) \quad [11].$$

$$\sigma_{RMs} = H \sqrt{ \frac{1 + \left(\frac{R_{dot}}{H} \right)^2}{2 \ln \left(\frac{2\pi N_s}{3600R} \right)} } \quad (7)$$

$$Warning_s = \sigma_{RMs} \times 5.22$$

$$, \quad N_s \quad 0.1 \quad , \quad R_{dot}$$

$$, \quad (8)$$

$$R_{dot} = 0.0927 \sqrt{gL} \quad (m/sec) \quad (8)$$

$$, \quad g \quad \text{가} \quad , \quad L$$

$$Warning_s = 5 \quad , \quad 5 \quad (7)$$

$$, \quad 5$$

4.2

Fig. 6 .
 initial setup window, sailing mode window,
 harbour mode window,
 bending moments window, data
 display mode window, gauge
 check mode window trend trend window

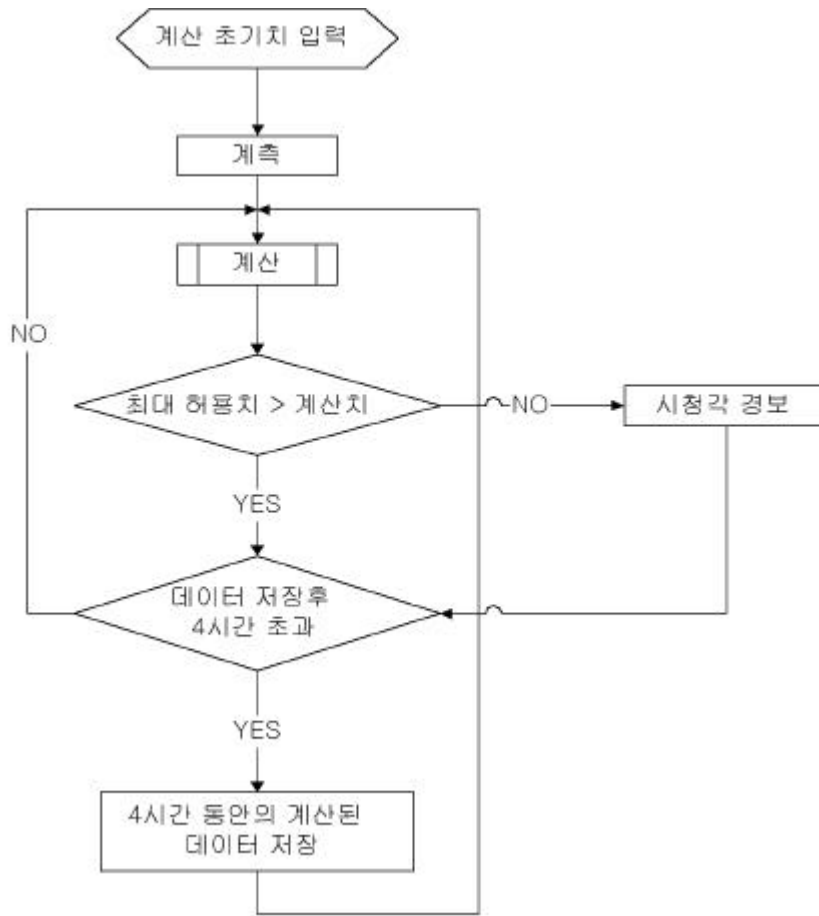


Fig. 6 A Flowchart of Program

windows 98 , visual
 basic 6.0 . OCX(OLE Custom
 Control) mhgaug32.ocx, mhhist32.ocx mhfram32.ocx ,
 mhgaug32.ocx , mhhist32.ocx , mhfram32.ocx

Fig. 7

Fig. 8

, Fig. 9

heeling trim

, Max , Min , Mean , PTP
 , S/D , Zero CP zero cross period .

. Fig. 10

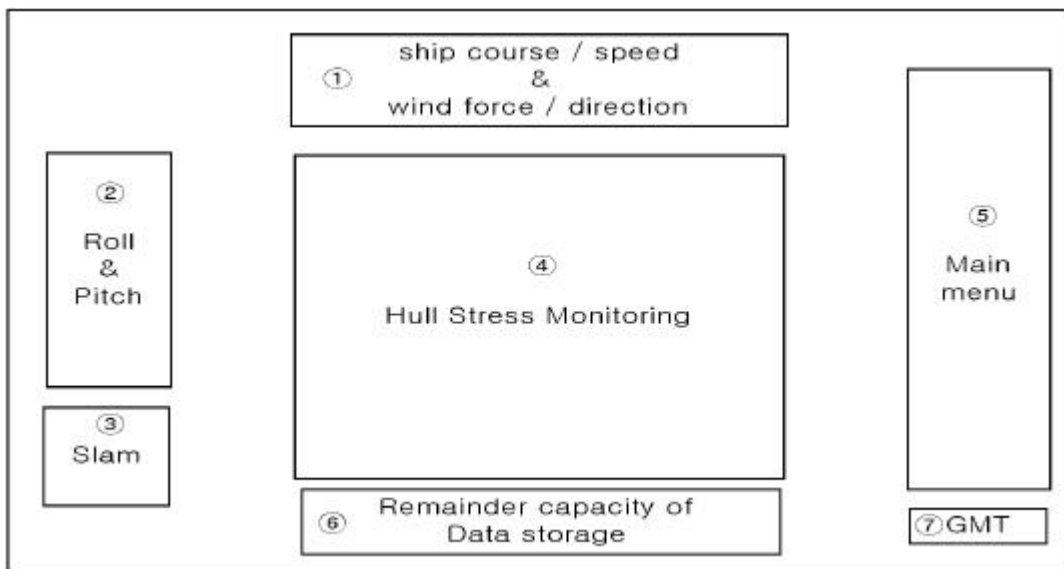


Fig. 7 Diagram of Main Display

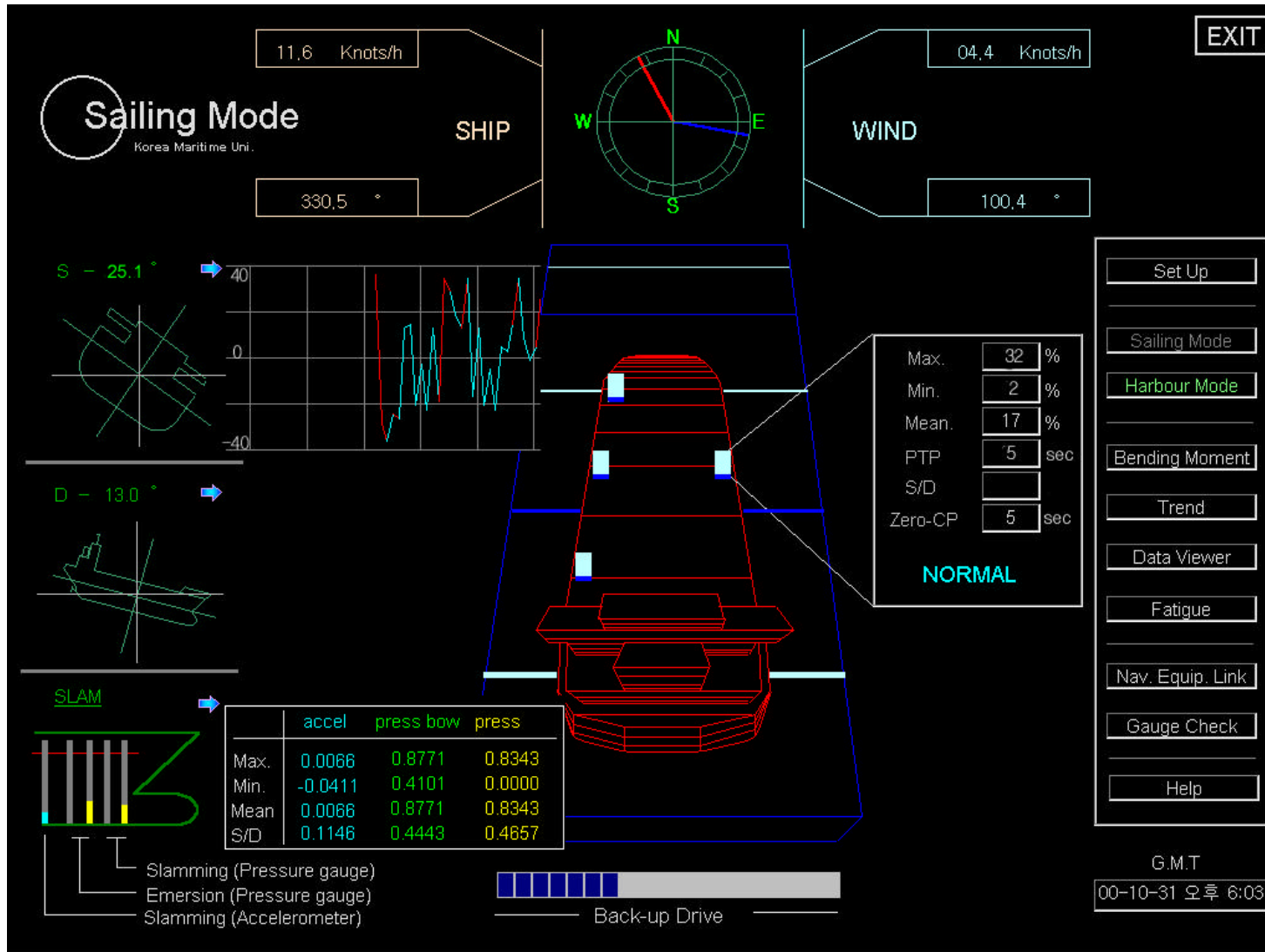


Fig. 8 Sailing display mode

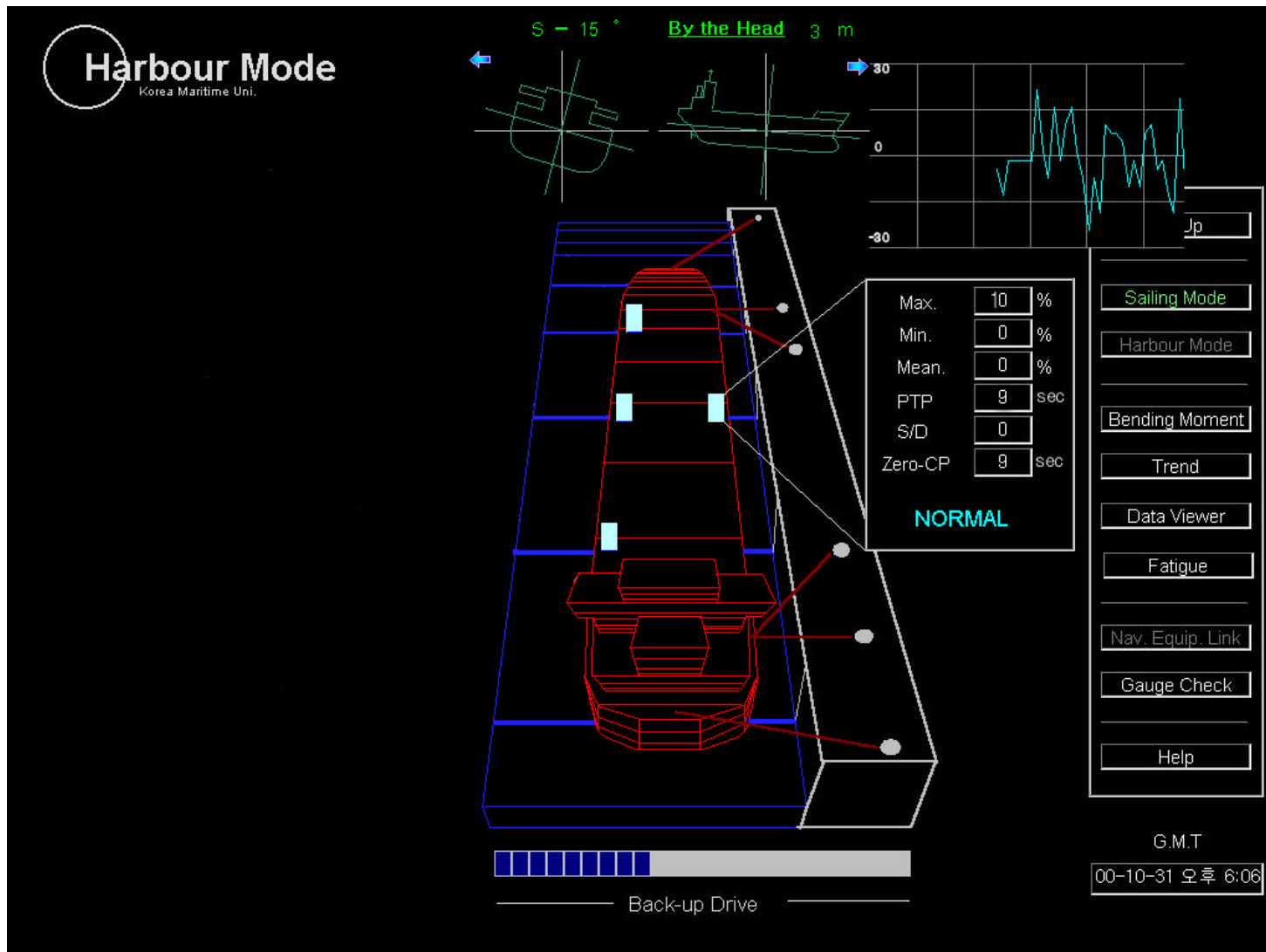
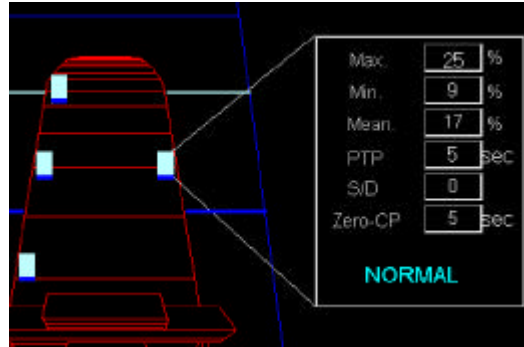
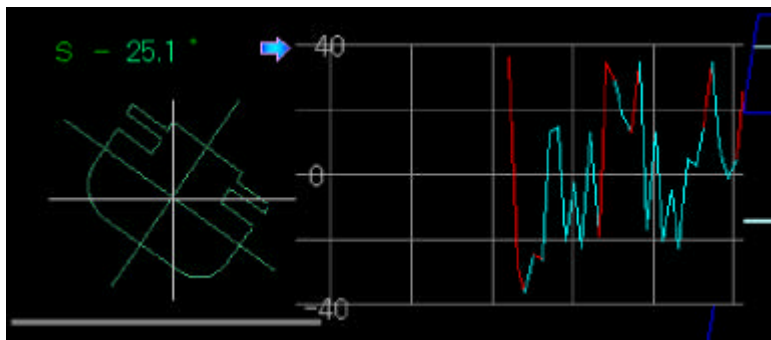


Fig. 9 Harbour display mode

Hull Stress



Rolling



Slamming

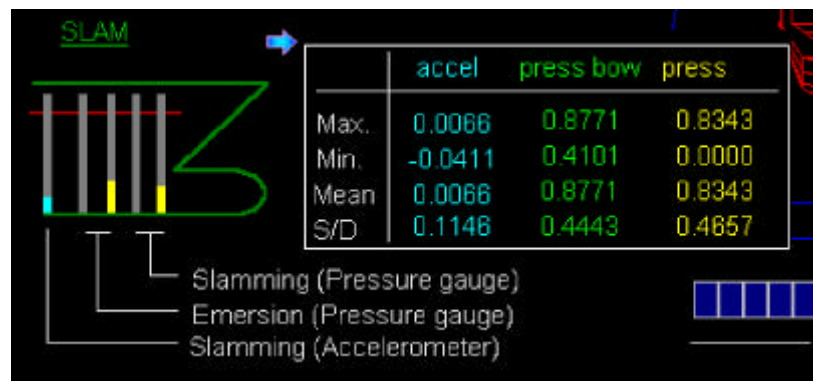


Fig. 10 Details of data display

Fig. 11

Ship's particulars
Principal Dimensions
LBSG
Warning

100

"Save"
"Cancel"

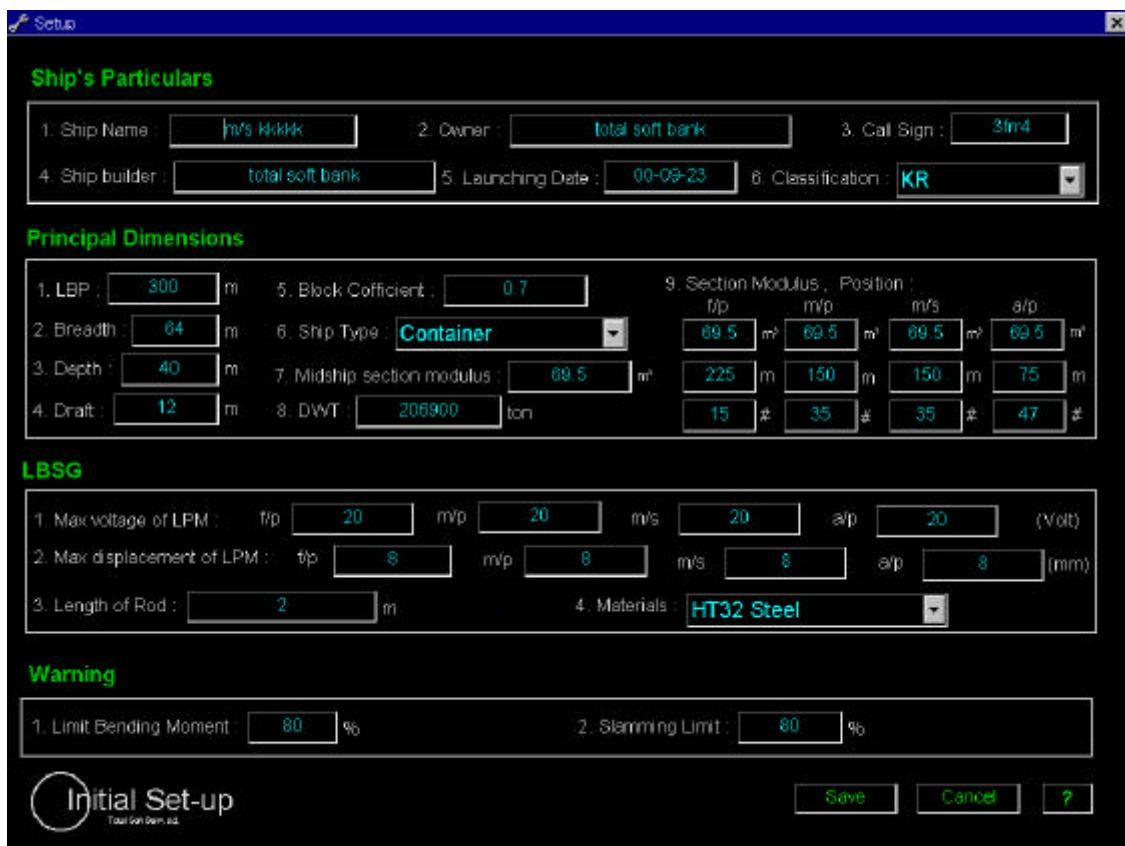


Fig. 11 Initial-Setup display

Fig. 12

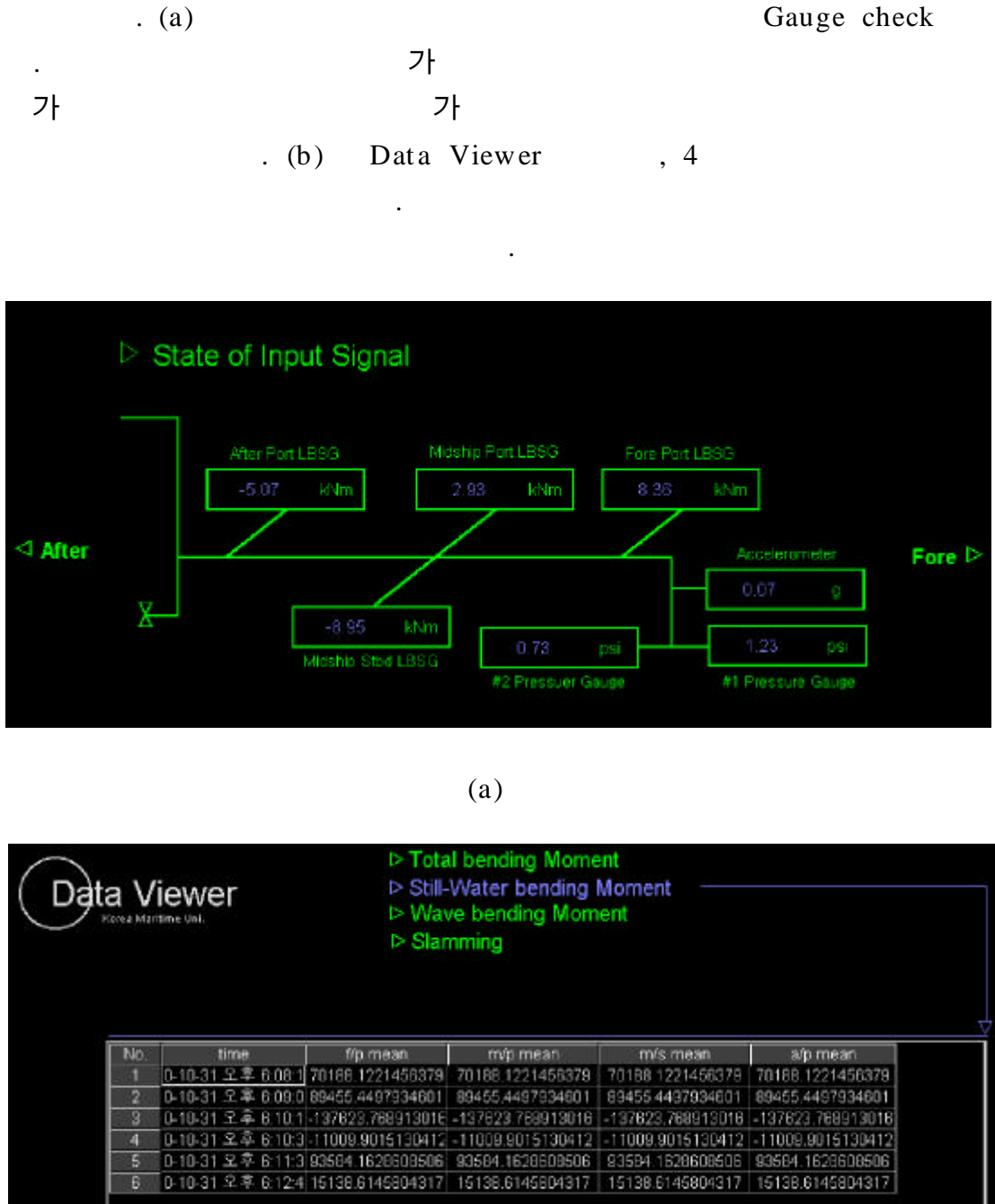


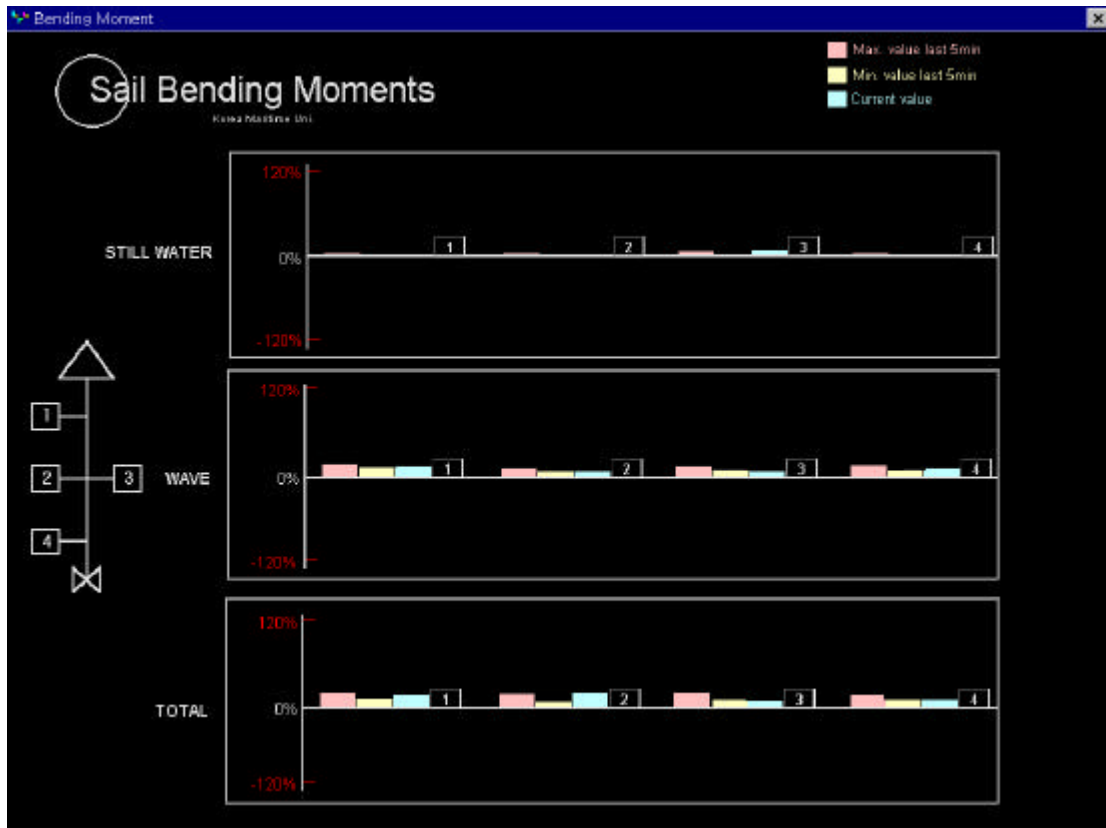
Fig. 12 Gauge check & Data viewer display

Fig. 13

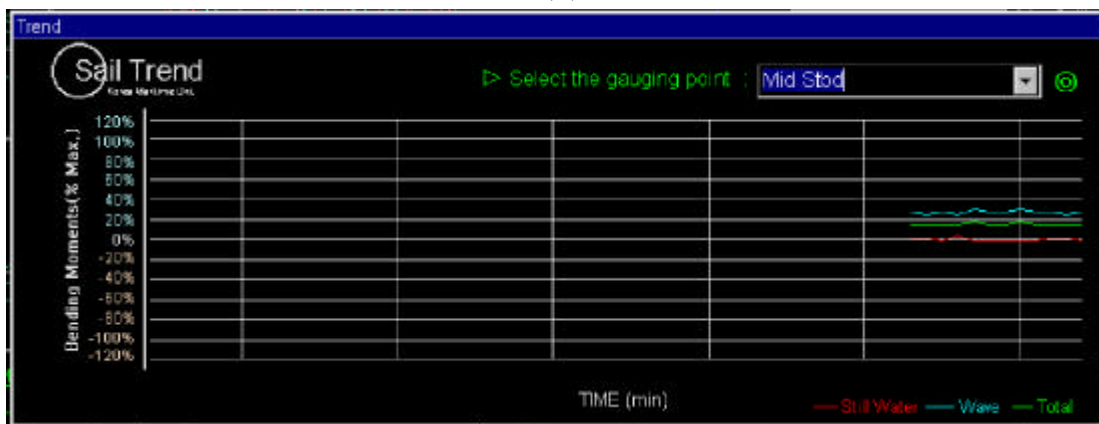
(a)

100

(b)



(a)



(b)

Fig. 13 Bar-graph & Trend display

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